

## **BLOWER**

### **BACKGROUND OF THE INVENTION**

#### **Field of the Invention**

The present invention relates to an improvement of a blower for use in various kinds of office automation apparatus, in particular, an improvement on thin type blower.

#### **The Related Art**

In an apparatus like an office automation apparatus which accommodates various kinds of electronic parts within a casing, heat generated by such electronic parts remains inside the casing which may cause the electronic parts to be broken. Therefore, the side wall of the casing is provided with a vent, in which a blower is mounted to discharge the heat in the case outside. The blower thus used has often been provided with two ball bearings in order to prevent the vibrations in both axial and radial directions to make the blower rotate stably.

Recently, the office automation apparatus has been developed to make it miniaturized. As the miniaturization develops, the blower itself is requested to be miniaturized in size and thickness. When the miniaturization in size and thickness is realized, a high precision bearing has to be used, since the allowable vibration of the rotary shaft supported by the bearings becomes narrower.

When high precision bearings are used to prevent vibration of a rotary shaft, it is often practiced to juxtapose two bearings (ball bearings) in a cylindrical bearing box. When juxtaposing two bearings, the vibration becomes small since the supporting span becomes large. However, when the supporting span is made long as described, the coaxiality of the two bearings has to be kept at a small rate, since otherwise a tension from an undesirable direction will effect the rotary shaft. It has been found that in general, the coaxiality of the two bearings to

be used in a blower for office automation apparatus has to be kept at 0.01 mm or less.

As a result, the coaxiality between the two bearings will be kept at a rate of 0.01 mm or less, however achieving this precision while inserting two bearings into a resin-made bearing box requires a high level of technique and is not feasible easily.

The present invention has been made in the light of this problem, and with a comparatively easy structure it is intended to provide a blower, in which its coaxiality of used bearings is made to be kept at 0.01 mm or less.

The present invention has been made to solve the above problem and according to a first aspect of the present invention, an impeller is fixed on one end of a shaft rotatably supported by bearings, where a ring-like magnet is provided inside the said impeller; characterized that on a blower causing said impeller and said shaft to rotate by the effect of a magnetic interference with the aforementioned magnet by supplying a current on a winding provided at a location relative to the said magnet of the stator, 2 ball bearings on which the width of the inner ring is smaller than the width of the outer ring, inserted into a bearing housing from one side, are used as aforementioned bearings.

Further, according to a second aspect of the present invention, an impeller is fixed on the end of a shaft rotatably supported by bearings, where a ring-like magnet is provided inside the said impeller; characterized that on a blower causing said impeller and said shaft to rotate by the magnet by supplying a current on a winding provided at a location relative to the said magnet of the stator, that 2 bearings, on which inner ring width is larger than the outer ring width are inserted into a bearing housing from one side, one of the inner race of the 2 bearings is pushed towards the outer bearing by a spring, are used as aforementioned bearings.

By the construction of the first aspect, since the width

dimension of the inner race inserted in the resin-made bearing box from one side thereof is smaller than the width of the outer race, if the outer race is contacted, the inner race is not, the two bearings become possible to be turned within a minute area at the contacting point as the turning center. Due to this turning, since the outer race and inner race can be displaced to the shaft, those are fixed at a position where the coaxiality is minimum.

Further, if constituted according to the second aspect, since one inner race of the two bearings is pushed toward the other bearing by a spring and a pressurization is given, it becomes possible to maintain the position where the coaxiality becomes minimum.

#### Brief description of the drawings

Fig. 1 is a vertically sectional view of the embodiment of the blower of the present invention.

Fig. 2 is an enlarged sectional view of the two ball bearings.

Fig. 3 is a front view partially cut away viewing in the right direction of Fig. 1.

#### Embodiment

Hereinafter, an embodiment of the present invention is explained based on attached drawings. Fig. 1 is a vertically sectional view of the whole of a blower 1. Fig. 2 is an enlarged view of the shaft and bearings of Fig. 1, and Fig. 3 is a front view partially cut away seeing in the right direction of Fig. 1.

A casing 2 is a flange 3 approximately square in front view and made of a resin with a cylindrical portion 4 in a unitary manner and adapted to be mounted at necessary portions with bolts through holes 5 provided on the flange 3. On the casing 2, a motor base 7 supported by four spokes 6 is provided in a unitary manner.

*Sub a1*

At the center portion of the motor base 7, as shown in Fig. 1, a cylindrical bearing box 7b the tip of which is oriented inside and having a bottom portion 7a is formed, and into the bearing box 7b, two ball bearings 8, 9 are inserted from the right side of Fig. 1 and abutted to the bottom 7a to hold the outer races 8a and 9a (see Fig. 2). At an outer end of the outer race of the ball bearing 9, a spacer 10 for preventing removal is provided. As to details of the ball bearing, these are explained later. At the center of the motor base 7, a shaft 11 (rotary shaft) is come through and supported with the inner races 8b and 9b of the ball bearings 8 and 9 rotatably.

At one end of the shaft 11, a retainer ring 12 is coupled and abuts to the inner race 9b of the ball bearing 9. And at the other end of the shaft 11, a bush 13 is coupled, and between the bush 13 and the inner race 8a of the ball bearing 8, a spring 14 (belleville spring) is interposed and gives a pressurization to the inner race 8b of the ball bearing 8 in the right direction in Fig. 1.

*Sub a2*

At the portion where the bush 13 of the shaft 11 is coupled, the center portion of the plate-like yoke 15 is mounted by pressing and sticking working, and on the inner peripheral portion a ring-like magnet 16 is mounted on by adhering.

Further, on the outer periphery of the yoke 15 an impeller (blades) 17 is mounted. The periphery of the impeller opposes to the cylindrical portion 4 of the casing 2. On the rear portion of the motor base 7, a PC (printed circuit) Board 18 is mounted where parts 19, 20 are mounted which serve as control circuit of the blower 1. And, on the rear surface of the PC board 18, a stator core 21 mounted on the bearing box 7b of the motor base 7 is positioned, and its periphery opposes to the magnet 16. On the stator core 21 a stator winding 22 is wound, and connected to the PC board 18.

*Sub a3*

Thus structured blower 1, when the stator core 21 is

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magnetized by being supplied with power through the electronic parts 19, 20 as a controlling circuit to the stator winding 22, due to the mutual magnetic function with the magnet 16, the shaft 11 is rotated with the magnet 16 and the yoke 15. Since the impeller 17 is mounted on the periphery of the yoke 15, it rotates in a unitary manner and between the casing 2 and the cylindrical portion 4 an oriented air is generated.

Based on Fig.2 ball bearings 8, 9 are explained. Although the ball bearings 8, 9 are formed in such a manner that balls 8c, 9c are interposed between outer races 8a, 9a and inner races 8b, 9b through the retainers 8d, 9d, different from ordinary ball bearings, the width dimension  $L_2$  of the inner races 8b, 9b is smaller than the width dimension  $L_1$  of the outer races 8a, 9a. Due to the difference of the width dimension described above, between the inner race 8b of the ball bearing 8 and the inner race 9b of the ball bearing 9, clearance  $L_3$  ( $L_3 = L_1 - L_2$ ) can be obtained. With this clearance  $L_3$  (e.g., 0.1~0.3 mm), in a condition of contacting to the shaft 11 two ball bearings 8, 9 can be displaced while accommodating their positions to the direction of the shaft 11, therefore, the coaxiality of the ball bearings is increased.

And, at the other end of the shaft 11, the bush 13 is coupled, and since, between the bush 13 and the inner race 8a of the ball bearing 8, the spring 14 is interposed and gives a pressurization to the inner race 8b of the ball bearing 8, the coaxiality of the ball bearings is kept by pressurizing the inner race 8b of the ball bearing 8 to the side of inner race 9b of the ball bearing 9.

As described above, the present invention relates to the blower wherein as a bearing, in the resin-made bearing box, two ball bearings the width dimension of an inner race of which is smaller than the width dimension of an outer race are inserted from one side, and to a shaft the two ball bearings are fixed at a position where the coaxiality is minimum. Moreover, by giving a

pressurization to one inner race of the two ball bearings toward the other ball bearing with a spring, the coaxiality of the ball bearings can be kept at a minimum.

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